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Abstract

A 2D parallel Euler code based on the space-time conservation element and solution element (CE/SE) method is validated by solving the benchmark problem 1 in Category 3 of the Third CAA Workshop [1]. This problem concerns the acoustic field generated by the interaction of a convected harmonic vortical gust with a single airfoil. Three gust frequencies, two gust configurations, and three airfoil geometries are considered. Numerical results at both near and far fields are presented and compared with the analytical solutions, a frequency-domain solver GUST3D solutions, and a time-domain high-order Discontinuous Spectral Element Method (DSEM) solutions. It is shown that the CE/SE solutions agree well with the GUST3D solution for the lowest frequency, while there are discrepancies between CE/SE and GUST3D solutions for higher frequencies. However, the CE/SE solution is in good agreement with the DSEM solution for these higher frequencies. It demonstrates that the CE/SE method can produce accurate results of CAA problems involving complex geometries by using unstructured meshes.

1. Introduction

The noise generated by the interaction of vortical disturbances originating upstream with propellers or turbomachinery blades has been of great interest in noise studies. A model problem regarding a two-dimensional vortical wave interaction with a single airfoil was posed as a benchmark problem in Category 3 of the Third CAA Workshop [1]. This problem involves complex geometries and flow phenomena including vorticity shedding and acoustics radiation. It has been solved by using frequency-domain solvers [2] and high-order accurate time-

domain methods that solves the full nonlinear Euler equations [3,4,5].

In this paper, this problem is solved numerically by solving the unsteady 2D Euler equations using the space-time conservation element and solution element (CE/SE) method [6,7]. The CE/SE method was originally developed by Chang in 1991 [8]. It applies flux conservation to finite space-time volumes, and achieves second-order accuracy in both space and time for uniform space-time meshes. Its salient properties are summarized briefly as follows. First, both local and global flux conservations are enforced in space and time instead of in space only. Second. all the dependent variables and their spatial derivatives are considered as individual unknowns to be solved for simultaneously at each grid point. Third, every CE/SE scheme is based upon a non-dissipative scheme with addition of fully controllable numerical dissipation. This results in very low numerical dissipation. Fourth, the flux-based specification of the CE/SE schemes gives rise in a natural fashion to a simple yet generally effective non-reflecting boundary condition, which is an important issue in CAA. Some practical advantages of the CE/SE method over the high order finite difference methods are summarized as follows: 1) it can use both structured and unstructured meshes in one single algorithm to handle complex geometries, 2) it can avoid singular points without using any special treatment, and 3) it has the most compact stencil, this leads to efficient parallel computing and easy implementation of boundary conditions.

A detailed description of this method, and accompanying analysis, can be found in [6–11]. Applications of this method to CAA problems reveal that, on mesh sizes used in practice, the result is comparable to those obtained using high order compact difference schemes, even though the current solver is only 2nd-order accurate [12-18].

In this paper, a version of the 2D CE/SE Euler code that is adapted for parallel computing, is used to compute the acoustic field generated by the interaction of a vortical gust with a single airfoil. Three gust frequencies, two gust configurations, and three

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airfoil geometries are considered. Numerical results at both near and far fields are presented and compared with the analytical or GUST3D or DSEM solutions.

In the following, the description of the benchmark problem is given first, which is followed by the solution procedure, initial and boundary conditions, numerical results and conclusions.

2. Problem Description

Consider a Joukowski airfoil or a flat plate with chord length c. The mean flow parameters at infinity are x-velocity U_{∞} , y-velocity $V_{\infty}=0$, mass density ρ_{∞} , static pressure p_{∞} , and Mach number $M_{\infty}=0.5$. Flow variables are non-dimensionalized by using U_{∞} , c/2, $c/(2U_{\infty})$, ρ_{∞} and $\rho_{\infty}U_{\infty}^2$ as the velocity, length, time, density and pressure scales, respectively [1]. As a result, the non-dimensionalized mean flow parameters at infinity are:

$$\overline{u}_{\infty} = 1, \ \overline{v}_{\infty} = 0, \ \overline{\rho}_{\infty} = 1, \ \overline{p}_{\infty} = 1/(\gamma M_{\infty}^2)$$
 (1)

Here a perfect gas with the specific heat ratio $\gamma=1.4$ is assumed. In this paper, we will consider the steady-state Euler mean flow solution with its non-dimensionalized x-velocity \overline{v} , y-velocity \overline{v} , mass density $\overline{\rho}$ and static pressure \overline{p} satisfying the following boundary conditions at infinity:

$$\overline{u} = \overline{u}_{\infty}, \ \overline{v} = \overline{v}_{\infty}, \ \overline{\rho} = \overline{\rho}_{\infty}, \ \overline{p} = \overline{p}_{\infty}$$
 (at infinity)

Also, we will consider the unsteady Euler flow solution with its non-dimensionalized x-velocity u, y-velocity v, mass density ρ and static pressure p satisfying the following boundary conditions at infinity:

$$u = \overline{u}_{\infty} + u', \ v = \overline{v}_{\infty} + v',$$

$$\rho = \overline{\rho}_{\infty} + \rho', \quad p = \overline{p}_{\infty} + p' \quad \text{(at infinity)}$$

where

$$u' \stackrel{\text{def}}{=} -(v_g k_y/|\mathbf{k}|) \cos(k_x x + k_y y - \omega t) \qquad (4)$$

$$v' \stackrel{\text{def}}{=} (v_g \, k_x / |\mathbf{k}|) \cos(k_x x + k_y y - \omega t) \tag{5}$$

and

$$\rho' \stackrel{\text{def}}{=} 0 \quad \text{and} \quad p' \stackrel{\text{def}}{=} 0$$
 (6)

In Eqs. (4) and (5), (i) v_g , k_x and k_y are given constants; (ii) $|\mathbf{k}|$ is the absolute value of $\mathbf{k} = (k_x, k_y)$, i.e.,

$$|\mathbf{k}| \stackrel{\text{def}}{=} \sqrt{k_x^2 + k_y^2} \tag{7a}$$

and (iii)

$$\omega = \overline{u}_{\infty} k_x \tag{7b}$$

Note that Eqs. (4), (5), (7a) and (7b) imply that

$$\frac{\partial u'}{\partial x} + \frac{\partial v'}{\partial y} = 0$$
, $\frac{\partial u'}{\partial t} + \frac{\partial u'}{\partial x} = 0$, and $\frac{\partial v'}{\partial t} + \frac{\partial v'}{\partial x} = 0$ (8)

As a result, (u',v',ρ',p') (which represents the undistorted gust) is a plane-wave solution to the linearized Euler equations in which $(\overline{u}_{\infty},\overline{v}_{\infty},\overline{\rho}_{\infty},\overline{p}_{\infty})$ is the constant unperturbed solution. However, because the steady-state solution $(\overline{u},\overline{v},\overline{\rho},\overline{p})$ may vary in space, generally (u',v',ρ',p') is not a solution to the linearized Euler equations in which $(\overline{u},\overline{v},\overline{\rho},\overline{p})$ is the unperturbed solution. In the following numerical simulations, both 1D and 2D gusts are considered. For the case of 1D gust, it is assumed that

$$v_q = 0.02, \quad k_x = 0.1, 1.0, 3.0, \quad \text{and} \quad k_u = 0 \quad (9)$$

On the other hand, for the case of 2D gust, it is assumed that

$$v_q = 0.02$$
, and $k_x = k_y = 0.1, 1.0, 3.0$ (10)

3. Solution Procedure

The unsteady solution (u, v, ρ, p) referred to in Sec. 2 satisfies the following 2D Euler equations:

$$\frac{\partial u_m}{\partial t} + \frac{\partial f_m}{\partial x} + \frac{\partial g_m}{\partial y} = 0, \qquad m = 1, 2, 3, 4 \quad (11)$$

Here

$$u_1 = \rho, \quad u_2 = \rho u, \quad u_3 = \rho v, \quad u_4 = E_t$$
 (12)

$$f_1 = \rho u, \ f_2 = \rho u^2 + p, \ f_3 = \rho uv, \ f_4 = (E_t + p)u$$
(13)

$$g_1 = \rho v, \ g_2 = \rho u v, \ g_3 = \rho v^2 + p, \ g_4 = (E_t + p)v$$
(14)

with
$$E_t = p/(\gamma - 1) + \rho(u^2 + v^2)/2$$
. To proceed, let

$$u^* \stackrel{\text{def}}{=} u - \overline{u} - u', \quad v^* \stackrel{\text{def}}{=} v - \overline{v} - v',$$

$$\rho^* \stackrel{\text{def}}{=} \rho - \overline{\rho} - \rho', \quad p^* \stackrel{\text{def}}{=} p - \overline{p} - p'$$
(15)

and

$$\hat{u} \stackrel{\text{def}}{=} \overline{u} + u^*, \ \hat{v} \stackrel{\text{def}}{=} \overline{v} + v^*, \ \hat{\rho} \stackrel{\text{def}}{=} \overline{\rho} + \rho^*, \ \hat{p} \stackrel{\text{def}}{=} \overline{p} + p^*$$
(16)

Because $(\overline{u}, \overline{v}, \overline{\rho}, \overline{p})$ and (u', v', ρ', p') , respectively, represent the steady-state solution and the undistorted gust, Eq. (15) implies that (u^*, v^*, ρ^*, p^*) represents the contribution to the unsteady solution due to the presence of acoustic waves and "gust distortion". Moreover, because Eqs. (15) and (16) imply that

$$u = \hat{u} + u', \ v = \hat{v} + v', \ \rho = \hat{\rho} + \rho', \ p = \hat{p} + p' \ (17)$$

with the aid of Eq. (3), it can be shown that, at infinity,

$$\hat{u} = \overline{u}_{\infty}, \ \hat{v} = \overline{v}_{\infty}, \ \hat{\rho} = \overline{\rho}_{\infty}, \ \hat{p} = \overline{p}_{\infty}$$
 (at infinity)
(18)

Next, for each m=1,2,3,4, let (i) \hat{u}_m , \hat{f}_m and \hat{g}_m be the values of u_m , f_m and g_m , respectively, when u, v, ρ and p assume the values of $\hat{u}, \hat{v}, \hat{\rho}$ and \hat{p} , respectively, and (ii) \overline{u}_m , \overline{f}_m and \overline{g}_m be the values of u_m , f_m and g_m , respectively, when u, v, ρ and p assume the values of $\overline{u}, \overline{v}, \overline{\rho}$ and \overline{p} , respectively. Then because $(\overline{u}, \overline{v}, \overline{\rho}, \overline{p})$ represents the steady-state solution, one has

$$\frac{\partial \overline{u}_m}{\partial t} = 0$$
, and $\frac{\partial \overline{f}_m}{\partial x} + \frac{\partial \overline{g}_m}{\partial y} = 0$ (19)

Moreover, let

$$s_1 \stackrel{\text{def}}{=} u' \frac{\partial \overline{u}_1}{\partial x} + v' \frac{\partial \overline{u}_1}{\partial y} \tag{20}$$

$$s_2 \stackrel{\text{def}}{=} u' \frac{\partial \overline{u}_2}{\partial x} + v' \frac{\partial \overline{u}_2}{\partial y} + \overline{u}_1 \frac{Du'}{\partial t}$$
 (21)

$$s_3 \stackrel{\text{def}}{=} u' \frac{\partial \overline{u}_3}{\partial x} + v' \frac{\partial \overline{u}_3}{\partial y} + \overline{u}_1 \frac{Dv'}{\partial t}$$
 (22)

$$s_4 \stackrel{\mathrm{def}}{=} u' \frac{\partial \overline{u}_4}{\partial x} + v' \frac{\partial \overline{u}_4}{\partial y} + \overline{u}_2 \frac{Du'}{\partial t} + \overline{u}_3 \frac{Dv'}{\partial t} \qquad (23)$$

with

$$\frac{Du'}{\partial t} \stackrel{\text{def}}{=} \frac{\partial u'}{\partial t} + \overline{u} \frac{\partial u'}{\partial x} + \overline{v} \frac{\partial u'}{\partial y}$$
 (24)

and

$$\frac{Dv'}{\partial t} \stackrel{\text{def}}{=} \frac{\partial v'}{\partial t} + \overline{u} \frac{\partial v'}{\partial x} + \overline{v} \frac{\partial v'}{\partial y}$$
 (25)

Then Eqs. (8), (12)–(17) and (19), and the fact that (u',v',ρ',p') and (u^*,v^*,ρ^*,p^*) are small perturbations of $(\overline{u},\overline{v},\overline{\rho},\overline{p})$ (Note: $v_g\ll 1$) imply that the relation

$$\frac{\partial u_m}{\partial t} + \frac{\partial f_m}{\partial x} + \frac{\partial g_m}{\partial y} = \frac{\partial \hat{u}_m}{\partial t} + \frac{\partial \hat{f}_m}{\partial x} + \frac{\partial \hat{g}_m}{\partial y} + s_m \quad (26)$$

where m = 1, 2, 3, 4, is accurate to the first order in v_q . Combining Eqs. (11) and (26), one has

$$\frac{\partial \hat{u}_m}{\partial t} + \frac{\partial \hat{f}_m}{\partial x} + \frac{\partial \hat{g}_m}{\partial y} = -s_m, \quad m = 1, 2, 3, 4 \quad (27)$$

The current numerical simulation includes the following steps: (i) evaluation of the steady-state solution \overline{u}_m (or equivalently $(\overline{u}, \overline{v}, \overline{\rho}, \overline{p})$); (ii) evaluation of s_m using Eqs. (4), (5) and (20)–(23); (iii) evaluation of \hat{u}_m (or equivalently $(\hat{u}, \hat{v}, \hat{\rho}, \hat{p})$) using Eq. (27); and (iv) evaluation of (u^*, v^*, ρ^*, p^*) using Eq. (16).

Note that, (i) because the convective distortion of the gust generates no acoustic waves, p^* represents the acoustic pressure generated by the interaction of the gust with the airfoil, and (ii) the solved problem is linear, therefore its solution can be superimposed.

4. Initial and Boundary Conditions

Here we specify the numerical initial and boundary conditions applied when solving Eq. (27) for \hat{u}_m . At t=0, the marching variables are specified using the steady-state solution. Moreover, at any point on the airfoil surface, it is assumed that the velocity (u,v) is tangent to the airfoil. By using Eq. (17), this implies that

$$n_x \hat{u} + n_y \hat{v} = -(n_x u' + n_y v') \tag{28}$$

where (n_x, n_y) is any unit vector normal to the airfoil at the point under consideration.

On the other hand, at any spatial mesh point on the outer boundary of the computational domain, the values of u_m^*, u_{mx}^* and u_{my}^* at the *n*th time level, respectively, are specified using the (n-1/2)th time level value of the corresponding variables at the interior spatial point immediately neighboring to it. It has been shown numerically that these boundary condition are non-reflecting in nature [7].

5. Numerical Results

In the following, numerical results will be presented for two gust configurations, three gust frequencies, and three airfoil geometries. A 2D parallel CE/SE Euler solver, with no slope-limiting ($\alpha=0$) and with added-numerical-dissipation ($\epsilon=0.5$)[9], is used in the computation. The numerical solution of the acoustic pressure non-dimensionalized by v_g is plotted in all figures. The RMS pressure on the airfoil surface, and the sound intensity at one and four chord lengths from the origin, are presented for all cases to examine both the near and far field solutions.

5.1 Flat-Plate Airfoil

For this case, the analytical and numerical solutions are independent of k_y . The obtained solutions are plotted in Figs. 1–3 for $k_x=0.1,1.0$, and 3.0, respectively. It can be seen that a fairly good agreement with the analytical solution is achieved for both low and high frequencies. For $k_x=1.0$, the CE/SE solution is slightly different from the analytical solution.

As an example, for the case $k_x = 3.0$, a computational domain of $-23 \le x, y \le 23$ with 921x921 uniformly distributed mesh points is used. The total number of triangular cells is 846400 and $\Delta t =$

T/120, with $T=2\pi/\omega$ being the time period of the gust. The solutions settle into a time-periodic pattern by t=15T(3600 marching steps). Note that, for the CE/SE method, a time period of $\Delta t/2$ is advanced by each marching step. The computation takes 0.5 hours walltime using 32 CPUs on SGI Origin 2000 system with 250 MHz MIPS R10000 processors.

5.2 Symmetric Airfoil

The symmetric Joukowski airfoil of 12% thickness is considered here. As shown in Fig. 4, the airfoil is surrounded by an unstructured mesh (Fig. 4(a)) which, in turn, is embedded in a structured mesh (Fig. 4(b)). The unstructured mesh is generated by using TRUMPET [19]. The mean pressure on the airfoil surface is plotted in Fig. 5 and compared with the potential code FLO36 solutions. The corresponding acoustic solutions are shown in Figs. 6– 8 for the 2D gust case at different frequencies. A reasonable agreement between CE/SE and GUST3D solutions is observed for $k_x = k_y = 0.1$, while there are large discrepancies for $k_x = k_y = 1.0$ and 3.0. However, the CE/SE solutions agree well with the DSEM solution for $k_x = k_y = 1.0$ and 3.0. In Figs. 9-11, the computed solutions for the 1D gust case are shown for different frequencies. For $k_x = 0.1$ and 1.0, the CE/SE solutions agree well with the GUST3D solution in the near field and have slight differences in the far field.

Note that, for the symmetric airfoil case, the DSEM results given in [5] do not provide (i) any 2D gust solutions with $k_x = k_y = 0.1$; (ii) the sound intensity at one chord length due to the 2D gust with either $k_x = k_y = 1.0$ or $k_x = k_y = 3.0$; and (iii) any 1D gust solutions. Also note that no GUST3D solutions are available for the 1D gust case with $k_x = 3.0$ and $k_y = 0$, and the 2D gust case with $k_x = k_y = 3.0$.

As an example, a computational domain of $-23 \le x, y \le 23$ is used for $k_x = k_y = 3.0$. An unstructured mesh with 7392 triangles is used in the region of $-2 \le x, y \le 2$ while a uniform structured mesh formed from 420000 triangles is used in the rest of the computational domain. The RMS pressure solution converges by t = 10T (14000 marching steps). The computation takes 1.5 hours walltime using 16 CPUs on SGI Origin 2000 system with 400 MHz MIPS R12000 processors.

5.3 Cambered Airfoil

The airfoil considered here has the same thickness as the symmetric airfoil but with a camber ratio of 0.02 and an angle of attack of 2°. The steady lift is

no longer zero and the flow field is more complex. A larger computational domain is necessary. As an example, a computational domain of $-40 \le x, y \le 40$ is used for the case $k_x = k_y = 1.0$. Among the 643744 triangular cells that fill the entire computational domain, 5344 cells are contained in the region of $-2 \le x, y \le 2$ (see Fig. 12). The numerical results are obtained assuming $\Delta t = T/1050$.

The computed mean pressure on the airfoil surface is plotted in Fig. 13 with the FLO36 solution, showing that the CE/SE solution on the upper surface is slightly under-predictive. For the two frequencies $k_x=0.1$ and 1.0, the unsteady solutions are plotted in Figs. 14–15 for the 2D gust case and in Figs. 16–17 for the 1D gust case, respectively. The CE/SE solutions are very close to the GUST3D solution for both 1D gust case and 2D gust case at $k_x=0.1$. However larger discrepancies are observed for $k_x=1.0$. The CE/SE solution is very similar to the DSEM solution for the 2D gust case at $k_x=1.0$.

The domain size study is performed for the cambered airfoil for the 2D gust case at $k_x=1.0$. The near field solutions, including the RMS pressure on the airfoil surface and the sound intensity at one chord length, obtained in a smaller domain of $-23 \le x, y \le 23$, are identical to the solution presented above. The sound intensity at four chord lengths is slightly different from the presented solution. The non-reflecting boundary condition has some reflections that are generated at the far-field boundary, which is also observed in the RMS pressure contour plot. However the reflection is generally less than 5% of the maximum value of the acoustic field and has little effect on the accuracy of the numerical solutions.

Grid refinement study is not performed because the mesh used in the computation is fine enough to capture both the acoustic waves and the gust. About 30–40 mesh points per wave length are necessary for the CE/SE method since it is 2nd order accurate. In the current computation, about 40 mesh points per wave length are used.

6. Conclusions

The 2D parallel CE/SE Euler code has been used to solve the Problem 1 in Category 3 of the Third CAA Workshop. Three gust frequencies, two gust configurations, and three airfoil geometries have been studied. A fairly good agreement between the CE/SE and GUST3D solutions in both near and far fields is achieved for the lowest frequency. There are some discrepancies between CE/SE and GUST3D solutions for higher frequencies. However, the CE/SE solution is in good agreement with the DSEM solution for these higher frequencies. It

is concluded that the CE/SE method can produce accurate solution with a simple and general non-reflecting boundary condition. No special treatment is needed for the singular geometric boundary points, such as the leading edge point of the flat-plate airfoil and the trailing edge point of the symmetric and cambered airfoils. The CE/SE method can handle the complex geometry well by using the unstructured mesh. This work demonstrates that the CE/SE method is capable of producing accurate solutions of CAA problems involving complex geometries.

Acknowledgments

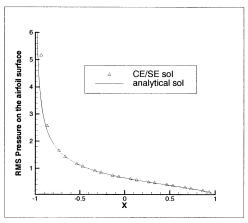
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7. References

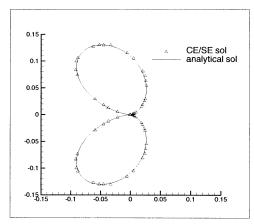
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(a) RMS pressure on the flat-plate surface.



(b) Sound intensity at one chord length.

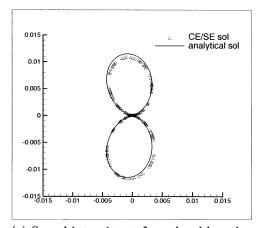
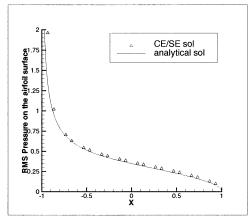
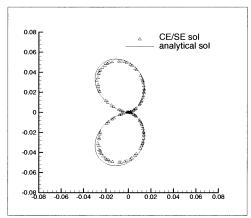


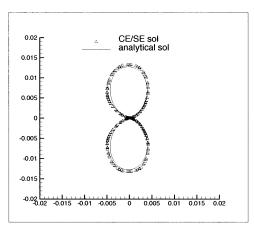
Figure 1: CE/SE solutions for the flat plate case assuming $k_x = 0.1$.



(a) RMS pressure on the flat-plate surface.

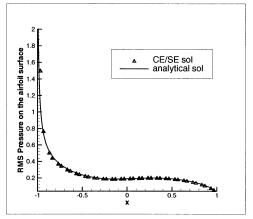


(b) Sound intensity at one chord length.

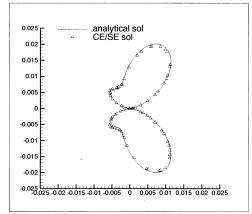


(c) Sound intensity at four chord lengths.

Figure 2: CE/SE solutions for the flat plate case assuming $k_x = 1.0$.



(a) RMS pressure on the flat-plate surface.



(b) Sound intensity at one chord length.

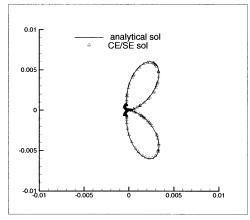


Figure 3: CE/SE solutions for the flat plate case assuming $k_x = 3.0$.

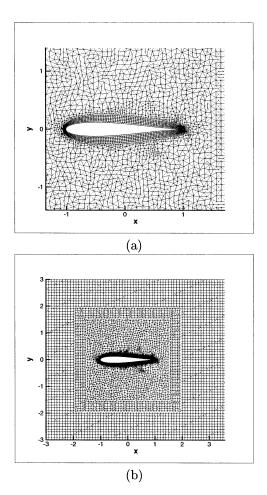


Figure 4: Structured/Unstructured mesh used for the symmetric airfoil calculation.

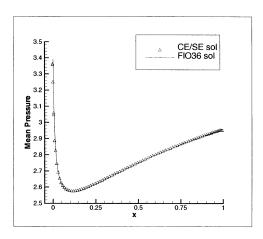
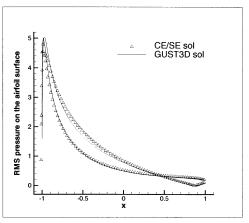
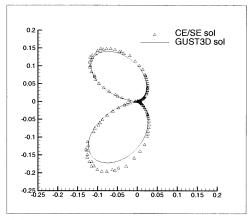


Figure 5: Mean pressure distribution on the airfoil surface for the symmetric airfoil.





(b) Sound intensity at one chord length.

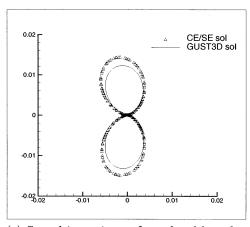
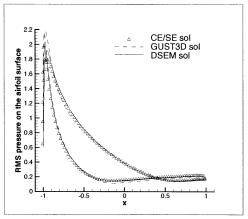
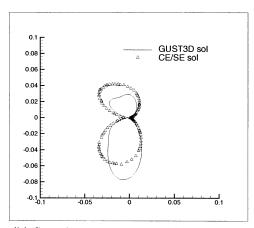
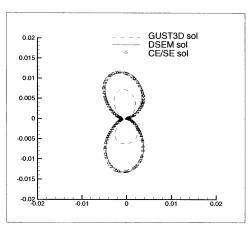


Figure 6: CE/SE solutions for the symmetric airfoil case assuming $k_x = k_y = 0.1$.



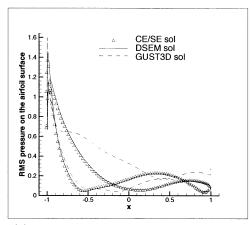


(b) Sound intensity at one chord length.

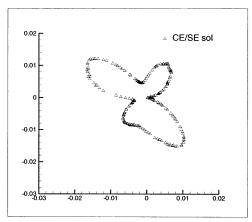


(c) Sound intensity at four chord lengths.

Figure 7: CE/SE solutions for the symmetric airfoil case assuming $k_x = k_y = 1.0$.



(a) RMS pressure on the airfoil surface.



(b) Sound intensity at one chord length.

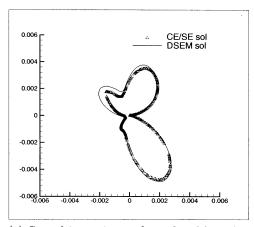
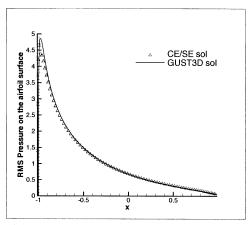
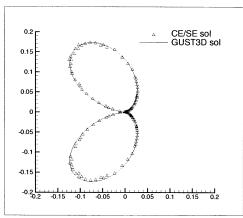
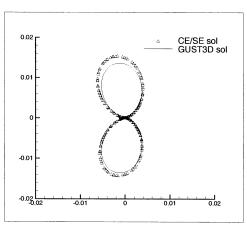


Figure 8: CE/SE solutions for the symmetric airfoil case assuming $k_x = k_y = 3.0$.



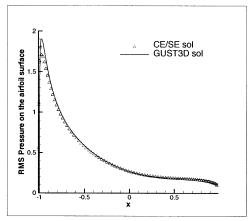


(b) Sound intensity at one chord length.

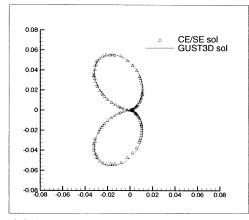


(c) Sound intensity at four chord lengths.

Figure 9: CE/SE solutions for the symmetric airfoil case assuming $k_x = 0.1, k_y = 0.0$.



(a) RMS pressure on the airfoil surface.



(b) Sound intensity at one chord length.

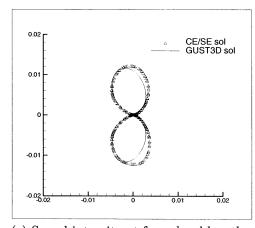
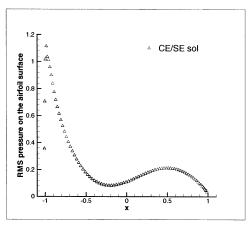
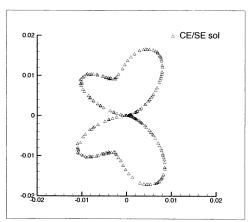
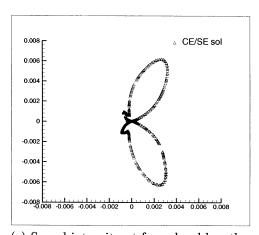


Figure 10: CE/SE solutions for the symmetric airfoil case assuming $k_x = 1.0, k_y = 0.0$.





(b) Sound intensity at one chord length.



(c) Sound intensity at four chord lengths.

Figure 11: CE/SE solutions for the symmetric airfoil case assuming $k_x = 3.0, k_y = 0.0$.

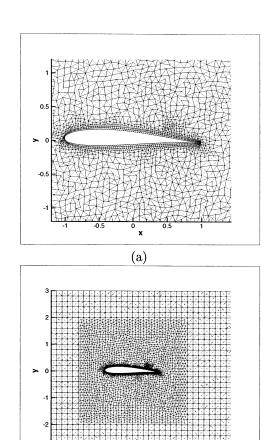


Figure 12: Structured/Unstructured mesh used in the cambered airfoil calculation.

(b)

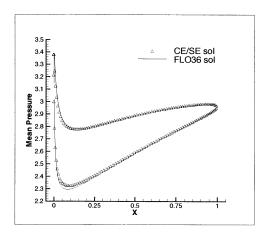
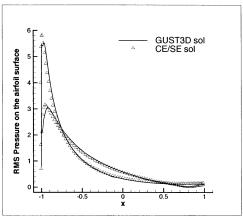
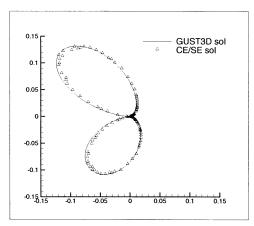
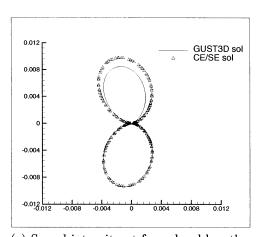


Figure 13: Mean pressure distribution on the airfoil surface for the cambered airfoil.



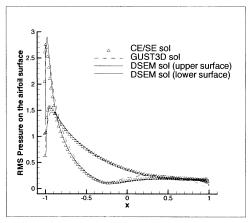


(b) Sound intensity at one chord length.

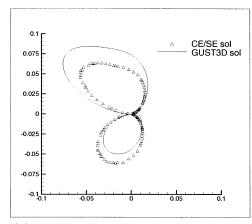


(c) Sound intensity at four chord lengths.

Figure 14: CE/SE solutions for the cambered airfoil case assuming $k_x = k_y = 0.1$.



(a) RMS pressure on the airfoil surface.



(b) Sound intensity at one chord length.

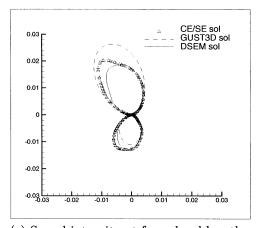
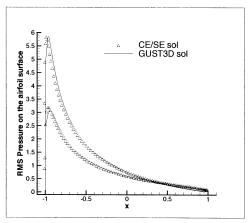
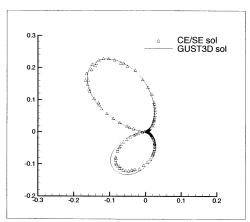
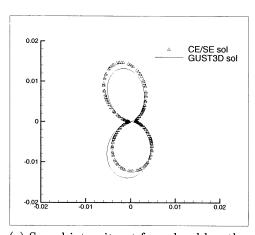


Figure 15: CE/SE solutions for the cambered airfoil case assuming $k_x = k_y = 1.0$.



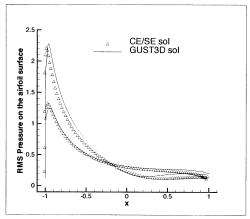


(b) Sound intensity at one chord length.

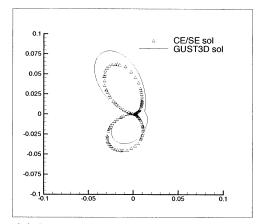


(c) Sound intensity at four chord lengths.

Figure 16: CE/SE solutions for the cambered airfoil case assuming $k_x = 0.1, k_y = 0.0$.



(a) RMS pressure on the airfoil surface.



(b) Sound intensity at one chord length.

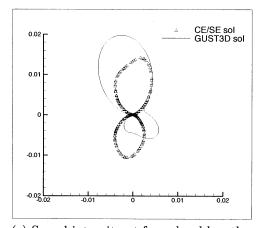


Figure 17: CE/SE solutions for the cambered airfoil case assuming $k_x = 1.0, k_y = 0.0$.

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13. ABSTRACT (Maximum 200 words)

A 2D parallel Euler code based on the space-time conservation element and solution element (CE/SE) method is validated by solving the benchmark problem I in Category 3 of the Third CAA Workshop [1]. This problem concerns the acoustic field generated by the interaction of a convected harmonic vortical gust with a single airfoil. Three gust frequencies, two gust configurations, and three airfoil geometries are considered. Numerical results at both near and far fields are presented and compared with the analytical solutions, a frequency-domain solver GUST3D solutions, and a time-domain high-order Discontinuous Spectral Element Method (DSEM) solutions. It is shown that the CE/SE solutions agree well with the GUST3D solution for the lowest frequency, while there are discrepancies between CE/SE and GUST3D solutions for higher frequencies. However, the CE/SE solution is in good agreement with the DSEM solution for these higher frequencies. It demonstrates that the CE/SE method can produce accurate results of CAA problems involving complex geometries by using unstructured meshes.

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